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Numerical analysis of wet plastic particle separation using a coupled DEM-SPH method

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ABSTRACT

The separation of different kind of plastic particles is required in the process of waste recycling. For the separation, drum processes with liquid can be used. The separation is based on the principle that particles either sink or float in a liquid depending on their densities. In this study, this process is numerically analysed for the separation of polyethylene terephthalate (PET) from polypropylene (PP) particles. The discrete element method coupled with the smoothed particle hydrodynamics method (DEM-SPH) is used for modelling purposes. The employment of the SPH for the modelling of the liquid exploits the strong side of this meshless method, namely, the relative ease in modelling large movements of the fluid with free surfaces and moving boundaries. This theoretical model is presented, and verification tests are performed, where a dam-break problem is considered as an example. Simulations of the plastic particle separation in the rotating drum are performed thereafter. The influences of the different operational and design parameters, such as the rotational velocity, feed rate, and number of lifters on the resultant purity of the plastic are estimated. It is expected that, in the future, the performed analysis will allow the numerical optimisation of drum separation processes.

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1. Introduction

Mechanical plastic recycling is currently one of the weakest steps in recycling because only a low percentage of plastic waste is reused compared to the amount of recovered metal, glass, and waste paper [\[1\].](#page-9-0) As the raw mixture of plastic waste usually includes various kinds of plastics (e.g., acrylonitrile-butadiene-styrene (ABS), polyethylene terephthalate (PET), polystyrene (PS), polyethylene (PE), polypropylene (PP), and polyvinyl chloride (PVC)), the separation process should classify waste into a number of reclaimable plastic fractions to meet the requirements that are needed in a high quality plastic recycling process for the purity and cleanliness of a polymer type [\[2\].](#page-9-1)

Mechanical plastic recycling processes can be divided into wet and dry separation techniques. Among the wet processes, drum separators based on the float-sink principle [\[3\]](#page-9-2) are widely used for separating granular materials [\(Fig. 1\)](#page-1-0). They are based on the fact that grains either sink or float in liquids depending on their densities. Before the separation process, the plastic wastes are shredded

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into small particle-like entities. The resulting plastic particle mixture is fed with water into a rotating drum. The sinking particles are lifted onto a sink launder and are then removed from the drum, while the floating particles are discharged out of the vessel with the water. To shift the cut point to a different density, it may be necessary to vary the density of the separating liquid. This is done either by dissolving materials of lower or greater density than water, for example alcohols or salts, or by suspending fine grained solids of greater density than water. The latter separating liquid is also known as a heavy medium and is extensively employed in the mineral processing industry [\[4\].](#page-9-3)

The successful separation of binary plastic mixtures in rotating drums was studied experimentally by Dodbiba et al. [\[1\].](#page-9-0) The same authors compiled a review of varying separation technologies and their efficiencies [\[5\].](#page-9-4) For multi-component plastic mixtures, a threestage process [\[6\]](#page-9-5) was proposed that results in good recovery rates for most kinds of plastic. Difficulties arise in float-sink separation if density differences between plastic fractions become low or in the case of elevated feed rates [\[7\].](#page-9-6)

Potentially, numerical modelling could contribute to the improvement of wet separation of particles. The discrete element method (DEM) can be used for modelling plastic particles. The DEM was first proposed by Cundall and Strack in 1979 and since then has been widely used in many areas of powder technology and

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Fig. 1. Scheme of a drum separator.

mechanical process engineering [\[8,9\].](#page-9-7) In the DEM, solid particles are separate entities, interacting with each other mostly by contact. This discrete character of the method allows a reduced set of constitutive assumptions to be used as compared to continuum approaches.

By modelling a mixed fluid-particle system, a coupling between solid particles and fluid is required. In the coupled system, the fluid phase can be modelled at the sub-particle level such that the momentum exchange between fluid and particles is resolved in detail [\[10\]](#page-9-8) or by using local averaging techniques [\[11\].](#page-9-9) The simulations at the sub-particle level allow a detailed analysis of interaction forces that act between the fluid and solid particles and therefore can be used for determination of drag correlations, for instance [\[12\].](#page-9-10) However, such simulations are computationally very expensive; therefore, they are usually limited to a small number of solid particles [\[13,14\].](#page-9-11) The use of local averaging techniques is computationally more efficient and allows simulations of much larger particle systems, while preserving the discrete characteristics of the particle flow.

For modelling the fluid flow, the Navier-Stokes equations are usually solved using mesh-based methods, like finite volume or finite element methods [\[15\].](#page-9-12) However, the application of mesh-based methods for modelling complex geometries and free surfaces, can be a challenging task [\[16\].](#page-9-13) To alleviate these difficulties, the smoothed particle hydrodynamics (SPH) method can be used. The SPH method, originally proposed by Gingold and Monaghan [\[17\]](#page-9-14) and Lucy [\[18\],](#page-9-15) is a meshless Lagrangian technique, which proved to be a suitable tool for modelling fluids in such areas as marine [\[19\],](#page-9-16) extrusion [\[20\],](#page-9-17) geophysical [\[21\]](#page-9-18) or coastal [\[22\]](#page-9-19) engineering. The major strength of this method is its meshless character, which makes the method very flexible and enables the simulation of engineering problems that might be difficult to capture by conventional mesh-based methods.

Recently, the two-way coupling between DEM and SPH based on a local averaging technique was proposed by Gao and Herbst [\[16\],](#page-9-13) Sun et al. [\[23\],](#page-9-20) and Robinson et al. [\[24\].](#page-9-21) The successful application of the two-way coupled DEM-SPH to slurry flow, abrasive wear, and magnetorheological fluids was demonstrated by Cleary [\[25\],](#page-9-22) Beck and Eberhard [\[26\],](#page-9-23) and Lagger et al. [\[27\],](#page-9-24) respectively. A detailed analysis of the sedimentation of one particle and a porous block presented in [\[24\]](#page-9-21) and a comparative study on coupling the DEM with mesh-based methods and DEM coupling with the meshless SPH method reported by Markauskas et al. [\[28\]](#page-9-25) support the conclusion that DEM-SPH is an appropriate and promising tool for modelling particle-laden fluid systems.

In the current investigation, the wet separation of plastic particles in a rotating drum is analysed numerically. The two-way coupled DEM-SPH method is used for this purpose. In our earlier work [\[28\]](#page-9-25) the study on the coupling was presented, while in the current investigation, the earlier developed framework is applied to a real engineering problem. The flexibility of the DEM-SPH method enables us to simulate rapid movements of the particle-laden fluid with free surfaces and moving walls and enables the analysis of the influence of various operational and design parameters on the separation process. To our best knowledge, the wet particle separation in a rotating drum is analysed numerically for the first time.

2. Governing equations

2.1. Governing equations of the solid phase

The DEM is used for modelling the solid particles. Newton's second law governs the motion of each particle:

$$
m_i \frac{d\mathbf{u}_i}{dt} = \mathbf{F}_i^c + \mathbf{F}_i^g + \mathbf{F}_i^{int}, \tag{1}
$$

where \mathbf{u}_i is the solid particle velocity, \mathbf{F}_i^c is the contact force, and \mathbf{F}_i^g is the gravity force. Moreover, \mathbf{F}_i^{int} is the interaction force acting between solid and fluid phases. The details regarding how this force is calculated are given in [Section 2.3.](#page-2-0) The contact force for particle P_i is obtained as a sum of all contact forces between P_i and particles in contact P*j*:

$$
\mathbf{F}_i^c = \sum_{j=1}^n \mathbf{F}_{ij}^c,\tag{2}
$$

where *n* is the number of contacts. For the calculation of the contact force in the normal direction, a Hertz contact model along with the damping model developed by Tsuji et al. (1992) [\[29\]](#page-9-26) are used. In the tangential direction, the force is described by a spring limited by the Coulomb law, characterised by the coefficient of tangential friction [\[30\].](#page-9-27) In the present study, only spherical particles are considered. A more detailed description of the DEM model can be found in [\[31,32\].](#page-9-28)

2.2. Governing equations of the fluid phase

The SPH method [\[17,18\]](#page-9-14) is used for modelling the fluid and is coupled with the DEM. The SPH treats the fluid in a completely meshfree fashion in terms of a set of sampling points (particles) [\[33\].](#page-9-29) The SPH particles represent a finite mass of the discretised fluid and carry information about all physical variables evaluated at their positions. Hydrodynamic equations for motion are derived for these particles.

The continuity equation and the momentum equation in a Lagrangian framework take the form [\[24\]:](#page-9-21)

$$
\frac{D\bar{\rho}_f}{Dt} + \nabla \cdot (\bar{\rho}_f \mathbf{u}_f) = 0, \tag{3}
$$

$$
\frac{D\bar{\rho}_f \mathbf{u}_f}{Dt} = -\nabla p + \nabla \cdot (\varepsilon \boldsymbol{\tau}) - \mathbf{f}^{\text{int}} + \bar{\rho}_f \mathbf{g},\tag{4}
$$

where $\bar{\rho}_f = \varepsilon \rho_f$ is the superficial density of the fluid, ε is the local mean fluid volume fraction, \mathbf{u}_f is the fluid velocity, p denotes the pressure, *t* is the viscous stress tensor, **f***int* is the interaction force between fluid and solid particles, and **g** is the gravitational constant.

The SPH particles carry variables, such as velocity, pressure, and mass. While no connectivity is modelled between the SPH particles, the function values are interpolated from the neighbouring particles using a smoothing kernel function. The kernel function is defined so that its value monotonously decreases as the distance between particles increases. The smoothing length *h* defines the influence radius of the kernel function. There are several kernel functions used in SPH, such as the Gaussian [\[34\],](#page-9-30) quadratic [\[35\],](#page-9-31) or quintic spline [\[36\].](#page-9-32) In the current study, a commonly used cubic spline kernel [\[37,38\]](#page-9-33) is utilised:

$$
W(r,h) = \alpha_D \begin{cases} 1 - \frac{3}{2}q^2 + \frac{3}{4}q^3, & 0 \le q < 1, \\ \frac{1}{4}(2-q)^3, & 1 \le q < 2, \\ 0, & q \ge 2, \end{cases}
$$
(5)

where $q = r/h$, $\alpha_D = 1/(\pi h^3)$ for the 3D case, *h* is the smoothing length, which defines the influence volume of the kernel, and *r* is the distance between the two points of interest.

An equation of state is used to estimate the pressure from the density field in the weakly compressible SPH method [\[33,39\]:](#page-9-29)

$$
p = \frac{\rho_0 c^2}{\gamma} \left[\left(\frac{\bar{\rho_f}}{\varepsilon \rho_0} \right)^{\gamma} - 1 \right],\tag{6}
$$

where ρ_0 is the initial density of the fluid phase, and *c* is the speed of sound. It is recommended to use $c = 10u$ to maintain the density variance to at most 1% [\[39,40\],](#page-9-34) where *u* is the maximum fluid velocity magnitude. The coefficient $\gamma = 7$ is commonly used in SPH. However, we experienced numerical instabilities when fluid with solid particles were considered in these simulations. Gao and Herbst [\[16\],](#page-9-13) following Morris et al. [\[40\],](#page-9-35) recommended using $\gamma = 1$ to avoid these numerical problems. Following this recommendation, $\gamma = 1$ is used in the present study.

The continuity Eq. [\(3\)](#page-1-1) in the SPH takes the form:

$$
\frac{D\bar{\rho}_a}{Dt} = \sum_b m_b \mathbf{u}_{ab} \cdot \nabla_a W_{ab},\tag{7}
$$

where indexes *a* and *b* indicate fluid particles, *m* is the mass, $\mathbf{u}_{ab} =$ **u**_{*a*} − **u**_{*b*} is the relative velocity between particles \mathcal{P}_a and \mathcal{P}_b , $\nabla_a W_{ab}$ = $\nabla_a W(r_a - r_b, h)$ is the gradient of the kernel function, and r_a and r_b are positions of the fluid particles P_a and P_b . The summation is performed over all neighbouring particles of particle P*a*.

The momentum conservation Eq. [\(4\)](#page-1-2) in SPH takes the form [\[40\]:](#page-9-35)

$$
\frac{D\mathbf{u}_a}{Dt} = -\sum_b m_b \left(\frac{p_a}{\bar{\rho}_a^2} + \frac{p_b}{\bar{\rho}_b^2}\right) \nabla_a W_{ab} + \mathbf{g} + \n+ \sum_b m_b \frac{\nu(\bar{\rho}_a + \bar{\rho}_b)}{\bar{\rho}_a \bar{\rho}_b} \cdot \frac{\mathbf{r}_{ab} \nabla_a W_{ab}}{|\mathbf{r}_{ab}|^2 + \delta^2} \mathbf{u}_{ab} + \frac{\mathbf{f}_a^{int}}{m_a}.
$$
\n(8)

The third term on the right-hand side in Eq. [\(8\)](#page-2-1) is a viscous term introduced by Morris [\[40\],](#page-9-35) where ν is the kinematic viscosity, and δ is a small number used just to keep the denominator non-zero, which is set to 0*.*1*h*.

Furthermore, \mathbf{f}_a^{int} in Eq. [\(8\)](#page-2-1) is the solid-fluid interaction force acting on the fluid particle \mathcal{P}_a due to the solid particles. The force \mathbf{f}_a^{int} is calculated as the sum over all solid particles in the domain of the fluid particle:

$$
\mathbf{f}_a^{\text{int}} = \sum_i -\frac{V_a W_{ai}}{\sum_b V_b W_{bi}} \mathbf{F}_i^{\text{int}}.\tag{9}
$$

where V_a is the volume of fluid particle, while \mathbf{F}_i^{int} is the interaction force acting on the solid particle (see Eq. [\(1\)\)](#page-1-3).

The fluid volume fraction ε_a of the fluid particle \mathcal{P}_a is calculated from the volumes of all solid particles P_i that are in the smoothing domain of the fluid particle P*a*:

$$
\varepsilon_a = 1 - \sum_i V_i W_{ai},\tag{10}
$$

where *V_i* is the volume of the solid particle P_i , while $W_{ai} = W(r_a - r_a)$ *ri*, *h*) is the kernel function Eq. [\(5\).](#page-2-2)

Fluid particles are moved using a velocity smoothed by the average in their neighbourhood according to the kernel function (i.e., XSPH variant introduced by Monaghan [\[41\]\)](#page-9-36):

$$
\frac{d\mathbf{r}}{dt} = \mathbf{v}_a + \varepsilon_{XSPH} \sum \frac{m_b}{\hat{\rho}_{ab}} (\mathbf{v}_b - \mathbf{v}_a) W_{ab},
$$
\n(11)

where ε_{XSPH} is the parameter that is equal to 0.5 and $\hat{\rho}_{ab} = (\rho_a + \rho_a)$ ρ_b)/2. This smoothed particle velocity reduces the fluid particle disorder, while it does not change the overall linear momentum.

An often-discussed topic in the SPH literature is the use of boundary conditions [\[42–44\].](#page-9-37) It is related to the fact that the description of boundaries in the SPH is not as straightforward as in other grid-based methods. There are several approaches to enforce no-penetration boundaries in the SPH, in which mostly special fluid wall particles are introduced. In our earlier study [\[28\],](#page-9-25) a modification of a no-slip no-penetration boundary was proposed, in which instantaneously generated ghost-fluid particles were used. These boundaries performed well in the test cases in [\[28\].](#page-9-25) However, in the current study, where particle separation in the rotating drum is simulated, we experienced numerical problems when the fluid particles are moved above the free surface of the fluid by the lifters [\(Fig. 1\)](#page-1-0). In this situation, a negative pressure (tension) in some of the fluid particles arose, which caused an artificial attraction between those particles and the nearby walls. To prevent this problem, a repulsive force boundary model proposed by Monaghan [\[45\]](#page-9-38) is used in the current study. For this boundary condition, the force between the fluid particle and the wall does not depend on the fluid particle pressure. Therefore, no artificial tension between a particle and a wall is generated. Gao et al. [\[16\]](#page-9-13) and Robinson et al. [\[24\]](#page-9-21) used the same type of boundary condition in their studies applying the coupled DEM-SPH method.

2.3. Fluid-solid interaction

The interaction force acting on a solid particle \mathbf{F}_i^{int} in this study is calculated as the sum of the drag force \mathbf{F}_i^D and the pressure gradient force $\mathbf{F}_i^{\nabla p}$:

$$
\mathbf{F}_i^{int} = \mathbf{F}_i^D + \mathbf{F}_i^{\nabla p}.\tag{12}
$$

In the current study, the correlation proposed by Di Felice [\[46\],](#page-9-39) which is well-anticipated in literature, is used for the calculation of the drag force:

$$
\mathbf{F}_{i}^{D} = \frac{1}{8} C_{d} \rho_{f} \pi d_{i}^{2} \left(\mathbf{u}_{f,i} - \mathbf{v}_{i} \right) \left| \mathbf{u}_{f,i} - \mathbf{v}_{i} \right| \varepsilon_{i}^{2 - \chi}, \qquad (13)
$$

where ε_i , d_i , \mathbf{u}_{fi} , and \mathbf{v}_i are the fluid fraction at the location of solid particle P_i , the solid particle diameter, the fluid velocity, and the solid particle velocity, correspondingly. The fluid fraction ε_i is obtained from the fluid fractions at the surrounding fluid particles:

$$
\varepsilon_{i} = \frac{\sum_{a} \varepsilon_{a} V_{a} W_{ai}}{\sum_{a} V_{a} W_{ai}}.
$$
\n(14)

The drag coefficient C_d and the coefficient χ are calculated as a function of the particle Reynolds number [\[46\].](#page-9-39)

Assuming that the pressure gradient ∇*p* arises only because of the interaction between solid particles and the fluid, F_i^D can be combined with $F_i^{\nabla p}$ [\[47\],](#page-9-40) which results in:

$$
\mathbf{F}_i^{int} = \frac{\mathbf{F}_i^D}{\varepsilon} - V_i \rho_f \mathbf{g}.
$$
 (15)

The force \mathbf{F}_i^{int} is used in Eq. [\(1\)](#page-1-3) and Eq. [\(9\).](#page-2-3) More details about this DEM-SPH model can be found in [\[28\].](#page-9-25)

3. Verification of the numerical model

For verification purposes of the theoretical model presented in [Section 2,](#page-1-4) numerical tests of a dam-break problem are performed. In [Subsection 3.1,](#page-3-0) a single-phase (liquid only) dam-break problem is addressed, while, in [Subsection 3.2](#page-3-1) a two-phase (liquid and solid particles) dam-break problem is simulated, and the results are compared with available results found in literature.

3.1. Dam break: single-phase test

In this test, a part of a rectangular container is filled with water [\(Fig. 2\)](#page-3-2). The width of the liquid column is $a = 0.2$ m and the height is $2a = 0.4$ m. The gravity force is acting downwards with the magnitude *a*, (i.e. 0*.*2m*/s*2). At the start of the simulation, an initialisation step is performed, during which the liquid particles reach a static equilibrium condition. Then, the right wall of the container is removed, and the liquid flows along the horizontal bottom plane. The 2D numerical test using the volume of fluid (VOF) method was initially performed and reported by Hirt and Nichols [\[48\].](#page-9-41) Sun et al. [\[23\]](#page-9-20) also used the same test. Two simulations are performed: one using an initial distance between liquid particles equal to *a/*10 and one using an initial distance between liquid particles equal to *a/*20. A total 3000 and 24,000 SPH particles are used for the first and second simulations, respectively. Liquid particle positions for the *a/*20 simulation at different time instances are shown in [Fig. 2.](#page-3-2) The position of the leading edge of the liquid vs time is presented in [Fig. 3,](#page-4-0) where SPH results performed with two different SPH particle resolutions with VOF results from [\[48\]](#page-9-41) can be compared.

[Fig. 3](#page-4-0) illustrates that, when using *a/*10 SPH particles, the difference between VOF and SPH results reaches 5% at $t\sqrt{2g/a} = 2.0$. The reduction of the size of the SPH particles by a factor of two (*a/*20) reduces this difference down to 1%. The results indicate that the used SPH model reproduces VOF results very well.

3.2. Dam break: two-phase test

A simulation of the dam-break problem, in which water is laden with solid particles, is performed as a second test case. The numerical results are compared with the experimental results reported by Sun et al. [\[23\].](#page-9-20) A rectangular container with size $20 \text{cm} \times 10 \text{cm} \times 15 \text{cm}$ (see [Fig. 4\)](#page-4-1) is divided into two sections (one larger and one smaller section) by a vertical wall. In the first step, solid particles are generated with fluid particles placed on top of them in the smaller section of the container [\(Fig. 4](#page-4-1) a). Spherical solid particles with diameter 2.7 mm and density 2500 kg/m³ are used for modelling of the glass beads. Moreover, 200 g of solid material, which results in 7762 particles, is generated. The Young's modulus of the particle equals 100 MPa. Its Poisson's ratio is 0.2. Its restitution coefficient is set to 0.9, and the friction coefficient equals 0.2, as reported for the solid particles in the experiment [\[23\].](#page-9-20) For modelling the water phase, which is used in [\[23\]](#page-9-20) in the performed experiment, 5870 SPH particles with a density of $\rho_0 = 1000 \text{kg/m}^3$, a dynamic viscosity

Fig. 2. SPH particles representing the liquid in the single-phase dam-break test.

Fig. 3. Position of the leading edge of the liquid in the single-phase dam-break test.

 $\mu = 0.001$ Pa·s, a smoothing radius $h = 5.4$ mm, and an initial distance between SPH particles equal to *h/*1*.*3 are used. An initialisation is performed during which the solid particles and the liquid settle by the action of gravity. In the actual simulation, the vertical wall (dam), which divides the container into two sections, is raised by a constant velocity of $v_x = 0.68$ m/s; therefore, the solid particle-fluid mixture is moving out from the filled section. The change of the position of the leading edge of the fluid and the solid particles is shown in [Fig. 5.](#page-5-0)

In [Fig. 5,](#page-5-0) the simulation results match well with the experiment. However, some minor divergence of the positions of the solid particles obtained numerically from the reported experimentally measured values at $t = 0.16 - 0.20$ s can be observed from the presented curves. The possible reason for this minor divergence could be the physical properties of solid particles (restitution coefficient and friction coefficient), which were roughly estimated in [\[23\].](#page-9-20)

4. Numerical analysis of wet plastic particle separation

4.1. Simulation setup and parameters

A numerical analysis of the separation of plastic particles using a rotating drum is performed. The outline of the drum is presented in [Fig. 1.](#page-1-0) During the separation process, the mixture of grains with water is fed into the rotating drum through the opening on the right side [\(Fig. 1a](#page-1-0)). By interaction of gravity and buoyancy forces, the grains with a density lower than the liquid density float, while the grains with a density higher than the liquid density start to sink. The floating grains with the liquid are discharged through the opening on the left side of the drum. The lifters attached to the walls of the drum lift the sunken particles and drop them on the sink launder.

The separation of PET from PP is simulated. The density of PET particles is 1350 kg/m 3 , and the density of PP particles is 950kg/m $^3\!;$ a restitution coefficient of 0.5 is used. A density of 1000 kg/m³ and a dynamic viscosity of 0*.*001Pa• s are used for the liquid aligned with the properties of water.

In the simulations, a simplified scheme of the laboratory scale drum shown in [Fig. 6](#page-5-1) is used. Initially, a prefill of the drum is numerically performed, where 0.0198m³ of water is generated inside of the drum and a simulation of 1 s is performed during which this water settles and partly flows out of the drum, forming a starting condition. Then, the mixture of solid particles and fluid is generated inside the drum in small chunks every 0.09 s. By varying the size of the chunk, a possible variation of the flow rate is achieved. In all performed simulations, the particle mixture contains an equal volume of PET and PP material. For modelling the PET and PP material,

spherical particles with random diameters between 3 and 4 mm are **Fig. 4.** SPH particles (representing the liquid) and solid particles in the two-phase spherical particles with random diameters between 3 and 4 mm are dam-break test.

Fig. 5. Temporal variation of the position of the leading edge of the the fluid and solid particles in the two-phase dam break test.

used. For modelling the water, SPH particles with a kernel length of $h = 8$ mm and an initial distance of $h/1.3$ are utilised. The kernel length is chosen based on the simulation results described in [\[24\]](#page-9-21) and the analysis of the settlement of a single particle in [\[28\].](#page-9-25) All simulations are performed for 40 s of operation time. During the simulation, the sinking/sunken solid particles are lifted from the bottom of the drum and are dropped into the sink remove zone (SRZ; see [Fig. 6\)](#page-5-1), where particles are removed from the simulation. The discharge process of the sinking/sunken solid particles is represented in a simplified manner, which has no implication on the accuracy of the simulation. The floating solid particles with the water flow through the opening in the drum and are also removed (float remove zone (FRZ) in [Fig. 6\)](#page-5-1).

Overall, 10 simulations are performed to analyse the sensitivity of the separation quality to different parameters. The considered parameters are given in [Table 1.](#page-5-2) As a basic setup (BS in [Table 1\)](#page-5-2) a simulation with a rotational velocity of the drum of 0.5*π*rad/s, four lifters, a solid feed rate of 29.3 g/s and a water flow rate of 2*.*07•10−3m3/s is utilised. Additionally, vertical walls inside the drum (see [Fig. 6\)](#page-5-1) are used to form a zone where the separation of solid particles should take place. Every simulation (S1-S9) differs from the BS by just one parameter with two times smaller and two times larger values being applied. In [Table 1,](#page-5-2) only the parameters for the BS and the parameters for S1-S9, which differ from the BS, are presented.

4.2. Simulation of basic setup

Several snapshots of the simulation of the BS at different instances of time are presented in [Fig. 7.](#page-6-0) In the figures, the SPH water particles are shown in a light blue colour and are semi-transparent.

Fig. 6. Scheme of the drum separator used in the simulations.

Table	
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Simulation parameters for testing of the drum separation.

The PET particles ($\rho=1350$ kg/m 3) are shown in a red colour, and the PP particles ($\rho = 950 \text{kg/m}^3$) are represented in dark blue. As mentioned in [Section 4.1,](#page-4-2) the drum is initially filled with water. The column of water is generated inside of the drum at $t = 0$ s [\(Fig. 7a](#page-6-0)), which settles down until 1 s [\(Fig. 7b](#page-6-0)). Then, the mixture of water and solid particles is generated inside the drum. In [Fig. 7c](#page-6-0) and [Fig. 7](#page-6-0) d, solid particles move up out of the liquid using the lifters. However, not only sunken PET particles (red colour) but also some water and floating PP particles (blue colour) are caught and transported to the SRZ (see [Fig. 6\)](#page-5-1). Here, the fluid and liquid particles are removed from the simulation, which has no effect on the separation process, as even in reality they would leave the system here.

The mass of the fed plastic particles and plastic particles inside the drum during the simulation are shown in [Fig. 8.](#page-6-1) While the shown mass of the fed particles in the figure is cut at 200 g, the particles are charged constantly until the end of the simulation. The mass of PET particles inside of the drum at first increases rapidly; however, from about $t = 6s$, the amount of PET particles inside remains almost unchanged. The stepwise character of the 'PET in drum' curve reflects the time intervals at which the rotating lifters remove the sunken PET particles. The amount of PP particles increases rapidly until about $t = 8$ s, but then a light steady increase forms, which remains until the end of the simulation.

From the amount of the removed particles, a mass flow rate is calculated. Because particles can be removed either when they are lifted to the SRZ, or when they float through the left opening in the drum (FRZ; see [Fig. 6\)](#page-5-1), two mass flow rates for every particle type are obtained and shown in [Fig. 9.](#page-6-2) In an ideal separation case, all PET particles should reach the SRZ, while all PP particles should exit through the FRZ. While no PET particle is transported to the FRZ, in the basic simulation case (BS) some PP particles move into the SRZ with the PET. Therefore, the resulting 'SRZ PP' curve is non-zero.

4.3. Analysis of the influence of different design and operational parameters

4.3.1. Influence of separating walls

The influence of separating vertical walls (see [Fig. 6\)](#page-5-1) is analysed. In many drum separators, they are used to confine the zone of the settling particles. Besides the BS simulation, in which vertical walls are used, a simulation without vertical walls (S1 in [Table 1\)](#page-5-2) is performed. The development of the mass flow rate of particle removal is shown in [Fig. 10.](#page-7-0) The 'S1 SRZ PET' and 'BS SRZ PET' curves are always kept at the same level during the simulation. However, the 'S1 SRZ PP' curve is always above the 'BS SRZ PP' curve, which indicates that, in the S1 case, more PP particles are lifted together with PET particles. In the last 15 s, the mass flow rate of PP particles reaching the SRZ increases in the simulation without the vertical walls ('S1 SRZ PP' curve); therefore, the mass flow rate of PP particles entering the FRZ decreases ('S1 FRZ PP' curve). It lets us conclude, that the use of the separating vertical walls helps to reduce the undesired mixing of the discharged PET particles with PP particles, which

Fig. 7. Simulation of particle separation in the rotating drum: snapshots taken at different time instances.

Fig. 8. Basic setup (BS) simulation: the mass of the generated particles and the mass of the particles inside of the drum.

makes the use of the separating vertical walls a preferable design solution.

4.3.2. Influence of the rotational velocity of the drum

A numerical analysis of the influence of the rotational velocity of the drum is conducted. In addition to the BS simulation, where the drum is rotated at 0.5π rad/s velocity, two simulations, where the drum rotates at 0.25πrad/s (simulation S2 in [Table 1\)](#page-5-2) and at 1.0πrad/s (S3), are performed. The mass flow rate of the resulting particle removal is shown in [Fig. 11.](#page-7-1) In all three simulations, the mass flow rate of PET particles removed through the SRZ is at the same level; however, in the S2 case (lowest rotational velocity), this level is reached a bit later than in the BS and S3 simulations. The 'BS SRZ PP', 'S2 SRZ PP' and 'S3 SRZ PP' curves show how many PP particles were lifted and removed together with the PET particles into the SRZ. In the S3 case, more PP particles are trapped together with the PET particles ('S3 SRZ PP' curve) than those that flow through the opening with the water ('S3 FRZ PP' curve).

A completely different situation can be seen in the S2 simulation; just a very few PP particles are lifted into the SRZ; therefore, the 'S2 SRZ PP' curve always remains at zero level. This is observed because, when the drum rotates more slowly, the PP and PET particles have more time to separate from each other, and PP particles are not caught together with PET particles. The 'BS SRZ PP' curve remains at an intermediate level between corresponding S2 and S3 results.

Fig. 9. Mass flow rate of the separated particles in the basic simulation (BS).

Fig. 10. Mass flow rate of particle removal for S1 and BS simulations.

Thus, the rotational drum velocity has a big influence on the resulting purity of the separated particles.

4.3.3. Influence of the number of lifters

In the BS simulation, four lifters (see [Fig. 6\)](#page-5-1) on the sides of the drum are used. Simulations with two (S4) and eight (S5) lifters are additionally performed. The results are presented in [Fig. 12.](#page-7-2) The 'BS SRZ PET' and 'S5 SRZ PET' curves are kept on the same level; however, the corresponding curve from the S4 simulation, where only two lifters are used, reaches this level only at about 35 s. This indicates that, in the S4 simulation, more PET particles are remaining inside the drum. Comparing the 'BS SRZ PP', 'S4 SRZ PP' and 'S5 SRZ PP' curves, it is clear that the lowest mass flow rate of PP particles removed with the PET is obtained when eight lifters are used. The 'S5 FRZ PP' tends to decrease during the second part of the simulation, which indicates that more PP particles remain in the drum.

4.3.4. Influence of the feed rate of solid particles

The influence of the feed rate of solid particles on the sorting process is analysed. Simulations with three different feed rates of solid particles are performed: BS with a feed rate of 29.3 g/s, S6 with a two times lower feed rate of 14.65 g/s, and S7 with a two times higher feed rate of 58.6 g/s. The results of the simulations are presented in [Fig. 13.](#page-7-3) Comparing the 'BS FRZ PP', 'S6 FRZ PP' and 'S7 FRZ PP' curves, the character of these curves is similar; however, the level of the mass flow rate indicates the differences in the feed rate. The curve

Fig. 11. Influence of the rotational velocity of the drum: the mass flow rate of particle removal.

Fig. 12. Influence of the number of lifters in the drum: the mass flow rate of particle removal.

'S6 FRZ PP' stays at about 4.5 g/s, and the curve 'BS FRZ PP' stays at about 9.0 g/s, while the curve 'S7 FRZ PP' stays at about 18.0 g/s. The influence on the PET particles coming to the SRZ can be seen from the comparison of the 'BS SRZ PET', 'S6 SRZ PET' and 'S7 SRZ PET' curves. While the 'BS SRZ PET' and 'S6 SRZ PET' curves are kept at a constant level from about 6 s, the 'S7 SRZ PET' curve increases until the end of the simulation. This is because the part of the lifter near the particle feed zone is fully loaded and therefore cannot lift all the sunken PET particles in simulation S7, as can be seen in [Fig. 14.](#page-8-0) With time, the amount of sunken PET particles in the drum increases, and the zone with the sunken PET particles increases too. Therefore, the bigger part of the lifter is used to lift the particles, which results in the increase of the rate of the removed PET particles ('S7 SRZ PET' curve). The differences in the 'BS SRZ PP', 'S6 SRZ PP', and 'S7 SRZ PP' curves indicate that the amount of PP particles removed along with the sunken PET particles increases when more particles are fed into the drum. However, the increase of these removed PP particles corresponds to the increase of the amount of the removed PET particles.

4.3.5. Influence of the water feed rate

The mixture of solid particles can be fed into the drum with different amounts of water. The influence of the water feed rate is analysed by performing three simulations with water feed rates of 1.035 l/s, 2.07 l/s, and 4.14 l/s. The results of these simulations are presented in [Fig. 15.](#page-8-1) The 'BS SRZ PET', 'S8 SRZ PET', and 'S9 SRZ PET' curves are at

Fig. 13. Mass flow rate of particle removal: influence of the feed rate of solid particles.

Fig. 14. Snapshots of the simulation S7.

the same level, which indicates that the amount of water makes no influence on the settling and removal of the PET particles. The small influence of the amount of the water can be observed by comparing the 'BS SRZ PP', 'S8 SRZ PP', and 'S9 SRZ PP' curves. The 'S9 SRZ PP' curve is below the other two curves, which indicates that, in S9, a bit less PP particles are caught with the settled PET particles.

Fig. 15. Mass flow rate of particle removal: influence of the water feed rate.

4.3.6. Comparison of resultant purity of PET

The resultant PET purity obtained in the simulations is summarised in [Fig. 16.](#page-8-2) Here, the purity is defined as the mass of PET particles removed in the SRZ divided by the mass of all particles that were removed in the SRZ:

$$
Purity_{PET} = \frac{mass_{PET,SRZ}}{(mass_{PET,SRZ} + mass_{PP,SRZ})}.
$$

As seen in [Fig. 16,](#page-8-2) the resultant purity for the BS is equal to 87.6%. The removal of the vertical walls (S1) reduces the purity to 74.0%. With the lowering of the rotational velocity of the drum the purity of PET increases to 100.0%, while the increase of rotational velocity reduces the purity to 69.0%. The reduction of the number of lifters from four to two (BS and S4 cases), increases the purity to 90.3%, while the increase of the number of the lifters (eight lifters in the S5 case) decreases the purity to 82.3%. It was found that, in the tested range, the feed rate of solid particles has just a small influence on the resultant purity (S6 and S7 cases). The purity of PET increased from 84.8% to 90.6% when the feed rate by water was increased from 1.035 l/s to 4.14 l/s (S8 and S9 cases). From all tested parameters, the rotational velocity has the biggest influence on the resultant purity of PET.

5. Conclusions

In the present study, an application of a coupled DEM-SPH scheme for the analysis of wet plastic particle separation was presented. The DEM-SPH scheme was described, and dam-break tests were performed. The results were compared with published results found in literature, which, together with our earlier study [\[28\],](#page-9-25) let us conclude that the validity of the technique is ensured. The numerical analysis of the PET particle separation from PP particles in a rotating drum was performed. The influence of different operational and design parameters, such as the rotational velocity, number of lifters, feed rate, was analysed. Numerical results show that the use of the separating vertical walls, lower rotational velocity, higher number of lifters, and higher water feed rate increases the purity of the separated particles. While the technique was validated by comparing the simulation results of a dam-break problem with published experimental data, a direct comparison of the numerical simulation of the drum separation process with experiments is a desirable and required next step in the future.

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